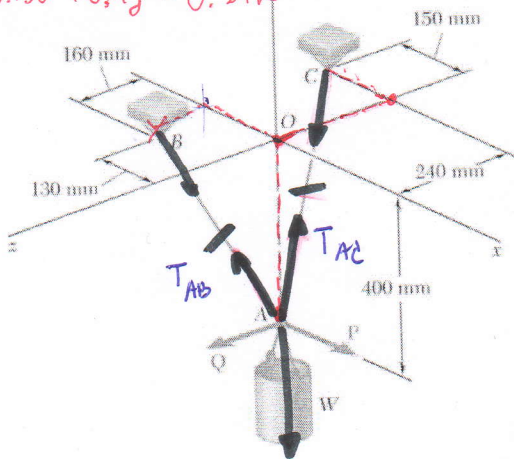


$$T_{AB} = T_{AC} = T$$

$$\vec{r}_{C/A} = 0.15\vec{i} + 0.4\vec{j} - 0.24\vec{k}$$



PROBLEM 2.121

A container of weight W is suspended from ring A . Cable BAC passes through the ring and is attached to fixed supports at B and C . Two forces $P = P\vec{i}$ and $Q = Q\vec{k}$ are applied to the ring to maintain the container in the position shown. Knowing that $W = 376 \text{ N}$, determine P and Q . (*Hint*: The tension is the same in both portions of cable BAC .)

$$\sum \vec{F} = 0 \Rightarrow \text{static equilibrium at } A$$

$$\vec{P} + \vec{Q} + \vec{T}_{AB} + \vec{T}_{AC} + \vec{W} = 0$$

$$\vec{W} = -376\vec{j}$$

$$\vec{P} = P\vec{i}$$

$$\vec{Q} = Q\vec{k}$$

$$\vec{i}: \sum F_x = 0$$

$$\vec{j}: \sum F_y = 0$$

$$\vec{k}: \sum F_z = 0$$

$$\vec{T}_{AB} = T_{AB} \vec{u}_{AB}$$

$$\vec{u}_{AB} = \frac{\vec{r}_{B/A}}{|\vec{r}_{B/A}|} \Rightarrow \vec{r}_{B/A} = -0.13\vec{i} + 0.4\vec{j} + 0.16\vec{k} *$$

$$|\vec{r}_{B/A}| = \sqrt{0.13^2 + 0.4^2 + 0.16^2} = 0.45$$

$$\vec{T}_{AB} = \frac{T_x [-0.13\vec{i} + 0.4\vec{j} + 0.16\vec{k}]}{0.45}$$

$$\vec{T}_{AB} = -0.29T\vec{i} + 0.89T\vec{j} + 0.356T\vec{k}$$

$$\vec{T}_{AC} = T_{AC} \vec{u}_{AC}$$

$$\vec{u}_{AC} = \frac{\vec{r}_{C/A}}{|\vec{r}_{C/A}|} \Rightarrow \vec{r}_{C/A} = -0.15\vec{i} + 0.4\vec{j} - 0.24\vec{k}$$

$$|\vec{r}_{C/A}| = \sqrt{0.15^2 + 0.4^2 + 0.24^2} = 0.49$$

$$\vec{T}_{AC} = \frac{T_x [-0.15\vec{i} + 0.4\vec{j} - 0.24\vec{k}]}{0.49}$$

$$\vec{T}_{AC} = -0.306 T_i + 0.816 T_j - 0.49 T_k$$

$$\Sigma \vec{F} = 0$$

$$\vec{P} + \vec{Q} + \vec{W} + \vec{T}_{AB} + \vec{T}_{AC} = 0$$

$$\left. \begin{aligned} P_i + Q_k - 376j - 0.29 T_i + 0.89 T_j + 0.356 T_k \\ - 0.306 T_i + 0.816 T_j - 0.49 T_k \end{aligned} \right\} = 0$$

$$\Sigma F_x = 0 \Rightarrow P - 0.29 T - 0.306 T = 0$$

$$P - 0.596 T = 0 \Rightarrow P = 0.596 \times 220.4$$

$$P = 131.36 \text{ N}$$

$$\Sigma F_y = 0 \Rightarrow -376 + 0.89 T + 0.816 T = 0$$

$$-376 + 1.706 T = 0$$

$$T = 220.4 \text{ N}$$

$$\Sigma F_z = 0 \Rightarrow Q + 0.356 T - 0.49 T = 0$$

$$Q - 0.134 T = 0$$

$$Q = 0.134 \times 220.4$$

$$Q = 29.53 \text{ N}$$



$$\vec{r} \rightarrow \vec{v} \rightarrow \vec{a} \rightarrow \vec{F} = m\vec{a}$$

PROBLEM 15.59

Knowing that at the instant shown the angular velocity of crank AB is 2.7 rad/s clockwise, determine (a) the angular velocity of link BD, (b) velocity of collar D, (c) the velocity of the midpoint of link BD.

$$\vec{\omega}_{BD}, \vec{v}_D, \vec{v}_G$$

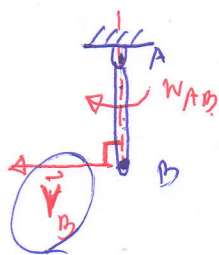
கிடைசியில் உள்ள Rigid Bodies கொடுக்க

Crank AB \Rightarrow Rotation about A

collar D \Rightarrow Translation

Link BD \Rightarrow General plane motion

① Crank AB (Rotation)



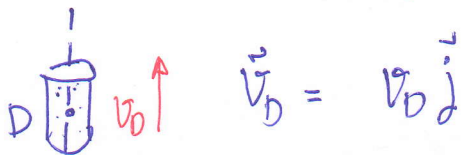
$$\vec{v}_B = \vec{\omega}_{AB} \times \vec{r}_{B/A}$$

$$= -2.7 \vec{k} \times [-0.125 \vec{j}]$$

$$\vec{v}_B = -0.3375 \vec{i}$$

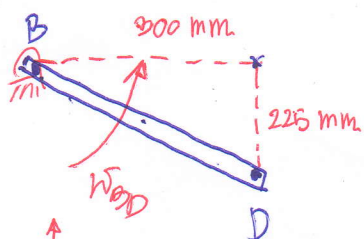
$$-\vec{k} \times \vec{i} = \vec{j}$$

② collar D (Translation)



கிடைசியில் உள்ள collar D
↑
↓
↑

③ Link BD (General plane motion)



$$\vec{v}_{GP} = \vec{v}_T + \vec{v}_R$$

$$\vec{v}_D = [\vec{v}_D]_{\text{Translation with B}} + [\vec{v}_D]_{\text{Rotation about B}}$$

$$\vec{v}_D = \vec{v}_B + \vec{v}_{D/B}$$

$$\vec{v}_D = \vec{v}_B + \vec{\omega}_{BD} \times \vec{r}_{D/B}$$

கிடைசியில் ω_{BD} ↑

$$\vec{v}_D \hat{j} = -0.9375 \hat{i} + \omega_{BD} \vec{k} \times [0.3 \hat{i} - 0.225 \hat{j}]$$

$$\vec{v}_D \hat{j} = -0.9375 \hat{i} + 0.3 \omega_{BD} \hat{j} + 0.225 \omega_{BD} \hat{i}$$

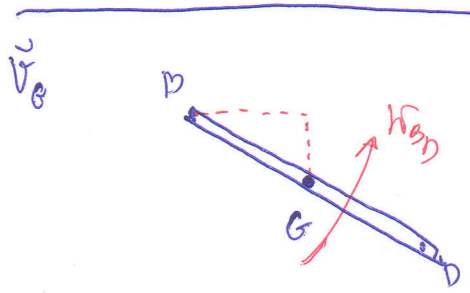
Wise ki direction se 2 vector ki magnitude ki direction i ka j

$$\hat{i} : 0 = -0.9375 + 0.225 \omega_{BD}$$

$$\omega_{BD} = 1.5 \text{ rad/s } \uparrow$$

$$\hat{j} : v_D = 0.3 \omega_{BD} = 0.3 \times 1.5$$

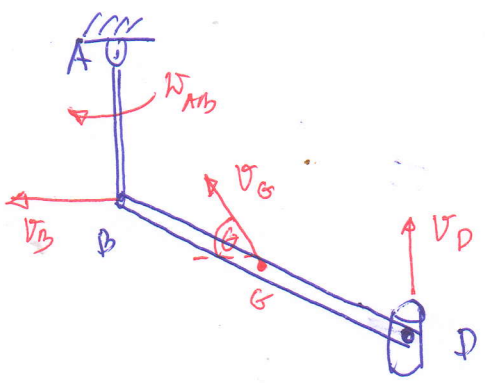
$$v_D = 0.45 \text{ m/s } \uparrow$$



$$\vec{v}_G = [\vec{v}_G]_{T \text{ with } B} + [\vec{v}_G]_{R \text{ about } B}$$

$$\vec{v}_G = \vec{v}_B + \vec{v}_{G/B}$$

$$\vec{v}_G = \vec{v}_B + \omega \times \vec{r}_{G/B}$$

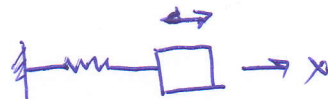


$$\vec{v}_G = -0.1687 \hat{i} + 0.225 \hat{j}$$

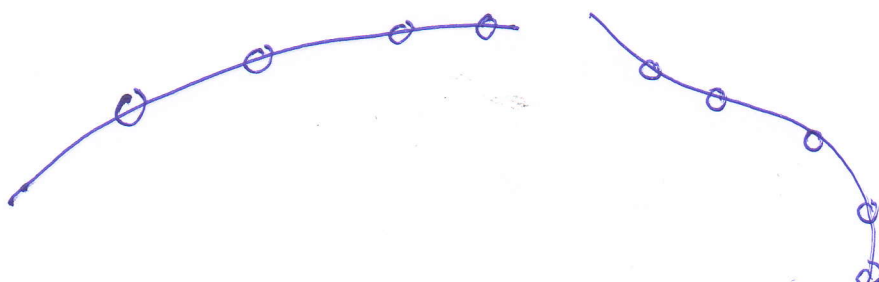
$$\theta = \tan^{-1} \left[\frac{0.225}{0.1687} \right] = 53.13^\circ$$

• गति (Motion)

1. Rectilinear motion



2. Curvilinear motion

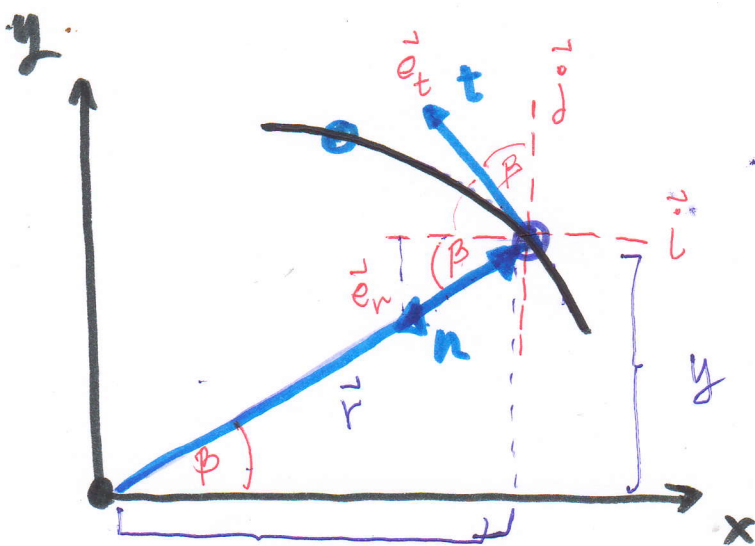


- Rectangular components (x, y) → Projectile
 - $\frac{dx}{dt} \Rightarrow v \hat{i}$
 - $\frac{dy}{dt} \Rightarrow v \sin \theta$
 - $a = -g$
- Normal & Tangential components (n, t)
- Radius & Transverse components (r, θ)

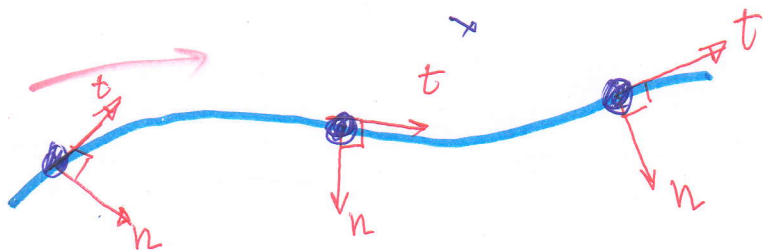
• Normal & Tangential components (n, t)

Unit vector \Rightarrow $x \Rightarrow \hat{i}$, $n \Rightarrow \frac{1}{r} \frac{dr}{dt} \hat{i} - \frac{v}{r} \hat{j}$
 $y \Rightarrow \hat{j}$, $t \Rightarrow \frac{v}{r} \hat{i} + \frac{dr}{dt} \hat{j}$

$|\hat{e}_n| = |\hat{e}_t| = 1$



$\vec{r} = x\hat{i} + y\hat{j}$



Acceleration : $\vec{a} = \frac{d\vec{v}}{dt} = \frac{d}{dt} [\rho \dot{\rho} \vec{e}_t]$

$$= \rho \frac{d}{dt} [\dot{\rho} \vec{e}_t]$$

$$= \rho \left[\dot{\rho} \frac{d\vec{e}_t}{dt} + \vec{e}_t \frac{d\dot{\rho}}{dt} \right]$$

$$= \rho \left[\dot{\rho} (\dot{\rho} \vec{e}_n) + \vec{e}_t \ddot{\rho} \right]$$

$$= \rho [\dot{\rho}^2 \vec{e}_n + \ddot{\rho} \vec{e}_t]$$

$$\vec{a} = \rho \dot{\rho}^2 \vec{e}_n + \rho \ddot{\rho} \vec{e}_t$$

$$\dot{\rho} = v$$

$\ddot{\rho} = \alpha = \text{Angular acceleration}$

$$\rho \dot{\rho}^2 = \rho \left[\frac{v}{\rho} \right]^2 = \frac{\rho v^2}{\rho^2} = \frac{v^2}{\rho}$$

$$\left. \begin{array}{l} v = \omega r \\ - \dot{\rho} \rho \\ \dot{\rho} = \frac{v}{\rho} \end{array} \right|$$

$$\vec{a} = \vec{a}_n + \vec{a}_t$$

$$\left. \vec{a} = a_x \vec{i} + a_y \vec{j} \right|$$

$$a_n = \rho \dot{\rho}^2 = \frac{v^2}{\rho}$$

$$a_t = \rho \ddot{\rho} = r \alpha$$

$$\vec{e}_n = -\cos\beta \vec{i} - \sin\beta \vec{j}$$

$$\vec{e}_t = -\sin\beta \vec{i} + \cos\beta \vec{j}$$

$$\begin{aligned} \frac{d\vec{e}_n}{dt} &= \sin\beta \frac{d\beta}{dt} \vec{i} - \cos\beta \frac{d\beta}{dt} \vec{j} \quad ; \quad \frac{d\beta}{dt} = \dot{\beta} \\ &= \dot{\beta} [\sin\beta \vec{i} - \cos\beta \vec{j}] \\ &= \dot{\beta} (-\vec{e}_t) \end{aligned}$$

$$\boxed{\frac{d\vec{e}_n}{dt} = \dot{\vec{e}}_n = -\dot{\beta} \vec{e}_t}$$

$$\begin{aligned} \frac{d\vec{e}_t}{dt} &= -\cos\beta \frac{d\beta}{dt} \vec{i} - \sin\beta \frac{d\beta}{dt} \vec{j} \\ &= \dot{\beta} [-\cos\beta \vec{i} - \sin\beta \vec{j}] \\ &= \dot{\beta} \vec{e}_n \end{aligned}$$

$$\boxed{\frac{d\vec{e}_t}{dt} = \dot{\vec{e}}_t = \dot{\beta} \vec{e}_n}$$

net

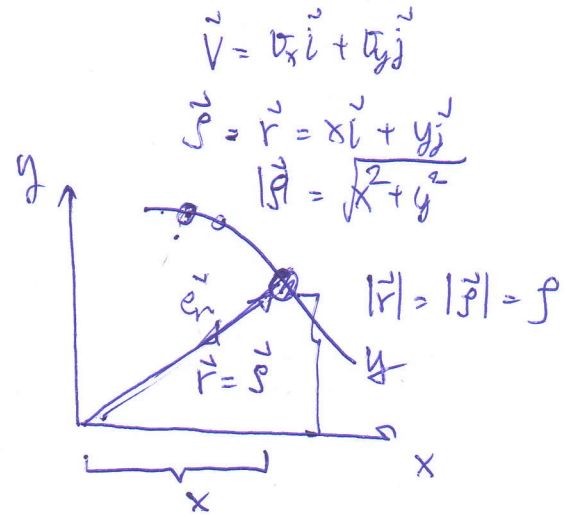
Position : $\vec{r} = -\rho \vec{e}_n$

Velocity : $\vec{v} = \frac{d\vec{r}}{dt} = \frac{d[-\rho \vec{e}_n]}{dt}$

$$\begin{aligned} &= - \left[\rho \frac{d\vec{e}_n}{dt} + \vec{e}_n \frac{d\rho}{dt} \right] \quad ; \quad \rho = \text{radius} \\ &= - \left[\rho (-\dot{\beta} \vec{e}_t) + 0 \right] \end{aligned}$$

radius is the distance from the center of curvature to the point on the curve
 radius of curvature

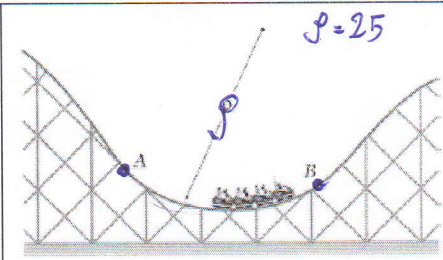
$$\begin{aligned} \vec{v} &= \rho \dot{\beta} \vec{e}_t \quad ; \quad \dot{\beta} = \text{Angular velocity} \Rightarrow \omega \\ \vec{v} &= \rho \dot{\beta} \vec{e}_t = \omega \rho \vec{e}_t \end{aligned}$$



$$a_n = 3g = 3 \times 9.81$$

$$\frac{d\rho}{dt} = 0$$

$$\rho = 25$$


PROBLEM 11.134

Determine the maximum speed that the cars of the roller-coaster can reach along the circular portion AB of the track if ρ is 25 m and the normal component of their acceleration cannot exceed 3 g.

Answer $(v_{\max})_{AB} = 97.6 \text{ km/h}$

$$a_n = \frac{v^2}{\rho}$$

$$; v = ?$$

$$\rho = 25$$

$$a_n = 3 \times 9.81 = 29.43$$

$$29.43 = \frac{v^2}{25}$$

$$v = 27.12 \text{ m/s}$$

$$v = 27.12 \frac{\text{m}}{\text{s}} = \frac{1 \text{ km}}{1000 \text{ m}} \times \frac{3600 \text{ s}}{1 \text{ hr}}$$

$$v = 97.632 \frac{\text{km}}{\text{hr}} \quad \#$$